AMENDMENTS TO THE SUBSTITUTE SPECIFICATION:

Page 4:

Please substitute the following paragraph for the paragraph beginning at line 14:

In the cross joint according to the present invention, preferably, an axial core side end surface of the contact portion is formed with a bottomed hole.

Thus, according to the present invention, the contact portion of the spider shaft is set larger than that of the central portion thereof, and hence the fitting between this contact portion and the rolling bearing can be set to have an interference (or a minute gap). Accordingly, even when vibrations are propagated from the wheels, the noises emitted due to the interference between the spider shaft and the rolling bearing can be reliably prevented by keeping uniform the minute gap between the central portion of the spider shaft and the rolling bearing.

Page 9:

Please substitute the following paragraph for the paragraph beginning at line 24:

Thus, the outside diameter of the contact portion 6a of the spider shaft 6 is set larger than that of the central portion 6b thereof, and hence the fitting between this contact portion 6a and the needle bearing 5 can be set to have an interference—(or a minute gap). Accordingly, even when vibrations are propagated from the wheels, noises emitted due to the interference between the spider shaft 6 and the needle bearing 5 can be reliably prevented by keeping uniform the minute gap_S between the central portion 6b of the spider shaft 6 and the needle bearing 5.

Page 11:

Please substitute the following paragraph for the paragraph beginning at line 8:

Note that an axial diameter of the spider shaft is approximately 10 mm, a depth of the bottomed hole 12 is 1.8 through 3.5 mm and preferably 2 through 3 mm, a radial thickness of the thin ring-shaped portion of the contact portion 6a is 0.6 to 1.2 mm, and a level difference (that is, a—the_minute gap S in FIG. 2 between the central

portion 6b and the rolling member 11) is 0.004 to 0.020 mm. Note that the minute gap indicated by the symbol S is illustrated in exaggeration.

Page 17:

Please substitute the following paragraph for the paragraph beginning at line 14:

As discussed above, according to the first through eighth embodiments of the present invention, the contact portion of the spider shaft sets its outside diameter larger than the outside diameter of the central portion thereof, and hence the fitting between this contact portion and the rolling bearing can be set to have the interference (or the minute gap). Accordingly, even when the vibrations are propagated, the noises emitted due to the interference therebetween can be reliably prevented by keeping uniform the a minute gap between the central portion of the spider shaft and the rolling bearing.

Please substitute the following paragraph for the paragraph beginning at line 26:

On the other hand, the contact portion of the spider shaft has the bottomed hole formed in the axial core

side end surface thereof and therefore takes on a comparatively thin ring shape. Accordingly, the contact portion is relatively small in its rigidity in the radial direction thereof and, even when the fitting between the contact portion and the rolling bearing is set to have the interference (or the minute gap), causes neither a large contact surface pressure nor a problem in terms of durability. Further, similarly, when assembling the cross joint to the yoke, the load for inserting (press-fitting) the rolling bearing (the needle bearing) is relatively small, and there is no real difficulty with the assembly process.

Page 20:

Please substitute the following paragraph for the paragraph beginning at line 25:

The roller 811 smoothly rotates owing to the oscillations of the spider shaft 806. However, however, the axial core of the roller 811 is not parallel but inclined to the axis of the spider shaft 806 in many cases, and the roller 811, with its rotation, moves in the axial direction within the cup 810.